

$$M_\beta := K_7 \cdot Q \cdot R_m \quad M_\beta = -2.223 \times 10^3 \text{ N} \cdot \text{m}$$

c) Circ. stress in shell, without stiffening rings

$$\begin{aligned} x_1 &:= 0.78 \sqrt{R_m \cdot t_{pv}} & x_1 &= 6.456 \text{ cm} & x_2 &:= x_1 & k &= 0.1 \\ \sigma_6 &:= \frac{-K_5 \cdot Q \cdot k}{t_{pv} \cdot (b + x_1 + x_2)} & \sigma_6 &= -3.104 \text{ MPa} \\ L < 8R_m &= 1 & L &= 1.6 \text{ m} & b_1 &= 14.411 \text{ cm} \\ \sigma_7 &:= \frac{-Q}{4t_{pv} \cdot (b + x_1 + x_2)} - \frac{12K_7 \cdot Q \cdot R_m}{L \cdot t_{pv}^2} & \sigma_7 &= 156.484 \text{ MPa} \end{aligned} \quad (4.15.25)$$

too high; we need a reinforcement plate of thickness;

$$t_r := 0.5t_{pv} \quad \text{strength ratio: } \eta := 1 \quad (4.15.29)$$

$$\sigma_{7r} := \frac{-Q}{4(t_{pv} + \eta \cdot t_r) \cdot b_1} - \frac{12K_7 \cdot Q \cdot R_m}{L \cdot (t_{pv} + \eta \cdot t_r)^2} \quad \sigma_{7r} = 67.28 \text{ MPa} \quad (4.15.28)$$

3) f) Acceptance Criteria

$$S = 1.379 \times 10^8 \text{ Pa} \quad S = 2 \times 10^4 \text{ psi}$$

$$|\sigma_{7r}| < 1.25S = 1$$

4) this section not applicable as $t_r > 2t_{pv} = 0$

4.15.3.6 - Saddle support, horizontal force given below must be resisted by low point of saddle (where height = h_s)

$$\begin{aligned} F_h &:= Q \cdot \left(\frac{1 + \cos(\beta) - 0.5 \cdot \sin(\beta)^2}{\pi - \beta + \beta \cdot \sin(\beta) \cos(\beta)} \right) & F_h &= 5.242 \times 10^4 \text{ N} & h_s &:= 9 \text{ cm} \\ \sigma_h &:= \frac{F_h}{b \cdot h_s} & \sigma_h &= 38.833 \text{ MPa} \end{aligned}$$

ANGEL Torispheric Head Design, using (2010 ASME PV Code Section VIII, div. 1, UG-32 Formed heads and sections, Pressure on Concave Side, Appendix 1-4 rules eq 3

$$P = 1.561 \times 10^6 \text{ Pa} \quad E = 1 \quad S = 2 \times 10^4 \text{ psi}$$

I.D.

$$D_i := 2R_{i_pv}$$

O.D.

$$D_o := D_i + 2t \quad D_o = 1.435 \text{ m}$$

Crown radius: Knuckle radius:

$$L_{cr} := 1D_i \quad L_{cr} = 1.36 \text{ m} \quad r_{kn} := 0.1D_i \quad r_{kn} = 0.136 \text{ m}$$

$$E = 1 \quad S_{div1} := 20000 \text{ psi}$$

Appendix 1-4 mandatory Supplemental Design Formulas

Ug-32 does not give equations for a range of crown and knuckle radii; these are found in App 1-4

$$\frac{L_{cr}}{r_{kn}} = 10$$

$$M := \frac{1}{4} \left(3 + \sqrt{\frac{L_{cr}}{r_{kn}}} \right) \quad M = 1.541$$

Minimum shell thickness:

$$t_{min} := \frac{P \cdot L_{cr} \cdot M}{2S \cdot E - 0.2P} \quad t_{min} = 11.871 \text{ mm} \quad (3)$$

note: we will need full weld efficiency for the above thickness to be permissible, as per UG-32(b)

this formula is only valid if the following equation is true (1-4(a))

$$\frac{t_{min}}{L_{cr}} \geq 0.002 = 1$$

$$\frac{t_{min}}{L_{cr}} = 8.729 \times 10^{-3}$$

ANGEL Torispheric Head Design, using (2010 ASME PV Code Section VIII, div. 2, part 4 rules)

2 nozzle head using standard dimension head

DRAFT

D. Shuman, LBNL, July12, 2011

4.3.6.1 Torispheric head with same crown and knuckle thickness, standard dimensions.

(a) Step 1, determine I.D. and assume the following:

thickness:

$$t := 9.30 \text{ mm} \quad t = 0.93 \text{ cm} \quad S_{\text{div2}} := 16700 \text{ psi} \quad 304L, 316L \text{ under div 2}$$

crown and knuckle radii:

$$L_{\text{cr}} = 1.36 \text{ m} \quad r_{\text{kn}} = 13.6 \text{ cm}$$

(b) Step 2- Compute the following ratios and check:

$$0.7 \leq \frac{L_{\text{cr}}}{D_i} \leq 1.0 = 1$$

$$\frac{r_{\text{kn}}}{D_i} \geq 0.06 = 1$$

$$20 \leq \frac{L_{\text{cr}}}{t} \leq 2000 = 1$$

for all true, continue, otherwise design using part 5 rules

(c) Step 3 calculate:

thickness, this is an iterated value after going through part 4.5.10.1 (openings) further down in the document

$$\beta_{\text{th}} := \arccos\left(\frac{0.5D_i - r_{\text{kn}}}{L_{\text{cr}} - r_{\text{kn}}}\right) \quad \beta_{\text{th}} = 1.11 \text{ rad}$$

$$\phi_{\text{th}} := \frac{\sqrt{L_{\text{cr}} \cdot t}}{r_{\text{kn}}} \quad \phi_{\text{th}} = 0.827 \text{ rad}$$

$$R_{\text{th}} := \begin{cases} \frac{0.5D_i - r_{\text{kn}}}{\cos(\beta_{\text{th}} - \phi_{\text{th}})} + r_{\text{kn}} & \text{if } \phi_{\text{th}} < \beta_{\text{th}} \\ 0.5D_i & \text{if } \phi_{\text{th}} \geq \beta_{\text{th}} \end{cases} \quad \phi_{\text{th}} < \beta_{\text{th}} = 1 \quad \phi_{\text{th}} \geq \beta_{\text{th}} = 0$$

$$R_{\text{th}} = 0.703 \text{ m}$$

(d) Step 4 compute:

$$C_{1ts} := \begin{cases} \left[9.31 \left(\frac{r_{\text{kn}}}{D_i} \right) - 0.086 \right] & \text{if } \frac{r_{\text{kn}}}{D_i} \leq 0.08 \\ \frac{r_{\text{kn}}}{D_i} \leq 0.08 = 0 & \end{cases} \quad (4.3.12)$$

$$\begin{cases} \left[0.692 \left(\frac{r_{\text{kn}}}{D_i} \right) + 0.605 \right] & \text{if } \frac{r_{\text{kn}}}{D_i} > 0.08 \\ \frac{r_{\text{kn}}}{D_i} > 0.08 = 1 & \end{cases} \quad (4.3.13)$$